

Version 1.0.10

**Beam**

Total length	L =	6.85	m
Number of elements	N =	100	

**Steel**

Young modulus	E =	210000	MPa
Poisson's coefficient	$\nu$ =	0.3	
Shear modulus	G =	80769	MPa

**Section - In Catalogue**

Selected Profile	=	IPE 400	
Weak flexural inertia	$I_z$ =	1317.8	cm <sup>4</sup>
Torsional constant	$I_t$ =	50.267	cm <sup>4</sup>
Warping constant	$I_w$ =	492149	cm <sup>6</sup>
Wagner factor	$\beta_z$ =	0	mm

**Lateral Restraints**

**Left End**

Position of Restraint /S	z =	0	mm
Lateral restraint	$v$ =	Fixed	
Torsional restraint	$\theta$ =	Fixed	
Flexural restraint	$v'$ =	Free	
Warping restraint	$\theta'$ =	Free	

**Right End**

Position of Restraint /S	z =	0	mm
Lateral restraint	$v$ =	Fixed	
Torsional restraint	$\theta$ =	Fixed	
Flexural restraint	$v'$ =	Free	
Warping restraint	$\theta'$ =	Free	

No intermediate lateral restraint

**Loading**

**Supports at Ends in the Plane of Bending**

Hinged at both ends

**External End Moments**

Left end moment	M1 =	0	kN.m
Right end moment	M2 =	-67	kN.m
End moments ratio (-M1/M2)	$\psi$ =	0.000	

**Distributed load**

Value at the origin	q1 =	-0.5	kN/m
Value at the end	q2 =	-0.5	kN/m
Abscissa/L at the origin	xf1 =	0	
Abscissa/L at the end	xf2 =	1	
Position /S	z =	200	mm

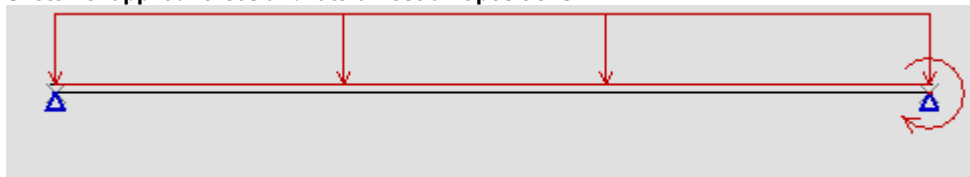
**Point load**

Value	F =	-52.5	kN
Abscissa/L	xf =	0.33	
Position /S	z =	200	mm

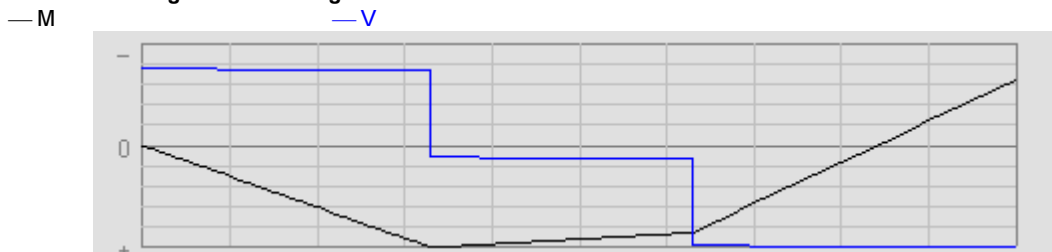
**Point load**

Value	F =	-52.5	kN
Abscissa/L	xf =	0.63	
Position /S	z =	200	mm

**Sketch of applied forces and lateral restraint positions**



**Bending and shear diagrams**



Maximum moment  $M_{max} = 103.91$  kN.m  
Abcissa/L  $x_f = 0.330$

### Critical Moment

#### Eigenvalue solving

Dichotomic process on determinant  
Convergence tolerance  $\epsilon = 0.0001$   
Number of iterations performed  $nit = 19$   
Convergence achieved  
Eigenvalue obtained  $\mu = 1.4868$

#### Critical Moment

Critical value of maximum moment  $M_{cr} = 154.49$  kN.m  
Abcissa/L  $x_f = 0.330$

#### Eigenmode

$v$

$\theta$

$v'$

$\theta'$

